

Static Failure Theories

5-1 → 5-7

- Fail:
- distortion
 - crack
 - rupture
 - yield
 - etc

Best way to know when a mechanical element fails:

Test!!

- exact same geometry
- same material
- same conditions

Expensive !!

→ human safety

→ high volume manufacturing

} these are good reasons to test

Given tensile, compressive, and/or shear strength for a given material, how do we choose what to check given the state of stress?

$$\sigma_x, \sigma_y, \sigma_z, \tau_{xy}, \tau_{yz}, \tau_{zx}$$

$$\sigma_1, \sigma_2, \sigma_3 \quad \text{or} \quad \tau_{1/2}, \tau_{2/3}, \tau_{1/3}$$

Theories are broken into two categories:

Ductile

$$\epsilon_f \geq 0.05$$

$$S_{yt} = S_{yc} = S_y$$

failure if
stress $> S_y$

?
 ϵ_f : strain at
ultimate failure

Brittle

$$\epsilon_f < 0.05$$

$$S_{ut}, S_{uc}$$

Ductile Failure

- Maximum Shear Stress (MSS)
- Distortion Energy Theory (DE)
- Ductile Coulomb-Mohr Theory (DCM)

Maximum Shear Stress Theory

Yielding begins when τ_{max} at any point in a element equals or exceeds the τ_{max} in a tensile test specimen of the same material, surface finish, ambient temperature, ..., when it begins to yield.



~~If $(\tau_{max})_{general}$~~

If $(\tau_{max})_{general} > (\tau_{max})_{tensile\ specimen}$

∴
failure

For the tensile specimen

$$\tau_{max} = \frac{S_y}{2}$$

3D state of stress:

$$\tau_{max} = \tau_{1/3} = \frac{\sigma_1 - \sigma_3}{2}$$

tensile yield strength

$$\therefore \tau_{1/3} \geq \frac{S_y}{2} \Rightarrow \sigma_1 - \sigma_3 \geq S_y$$

$$\sigma_1 - \sigma_3 = \frac{S_y}{n_{MSS}} \Rightarrow$$

$$n_{MSS} = \frac{S_y}{\sigma_1 - \sigma_3}$$

2D plane stress

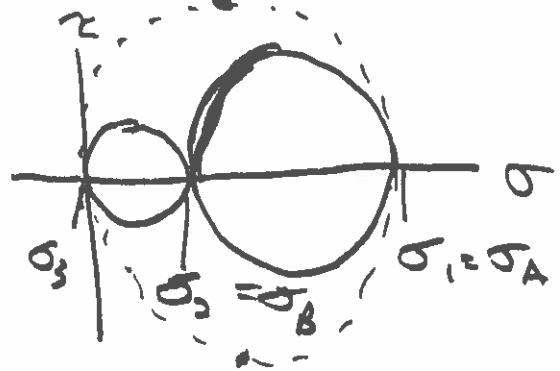
- one ~~part~~ principal stress is zero

$$\sigma_A, \sigma_B \Rightarrow \sigma_A \geq \sigma_B$$

principal stresses

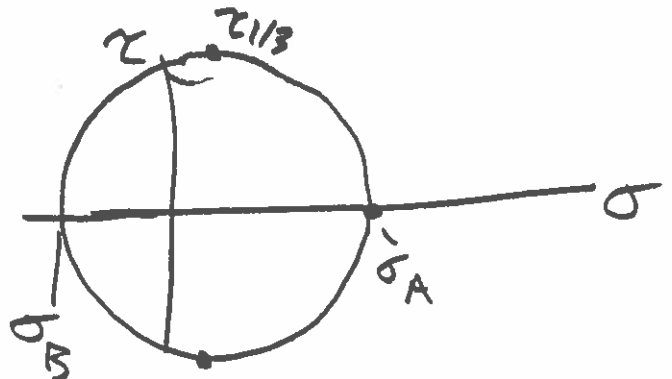
1) $\sigma_A \geq \sigma_B \geq 0$

$$\sigma_A \geq S_y \Rightarrow \text{failure}$$



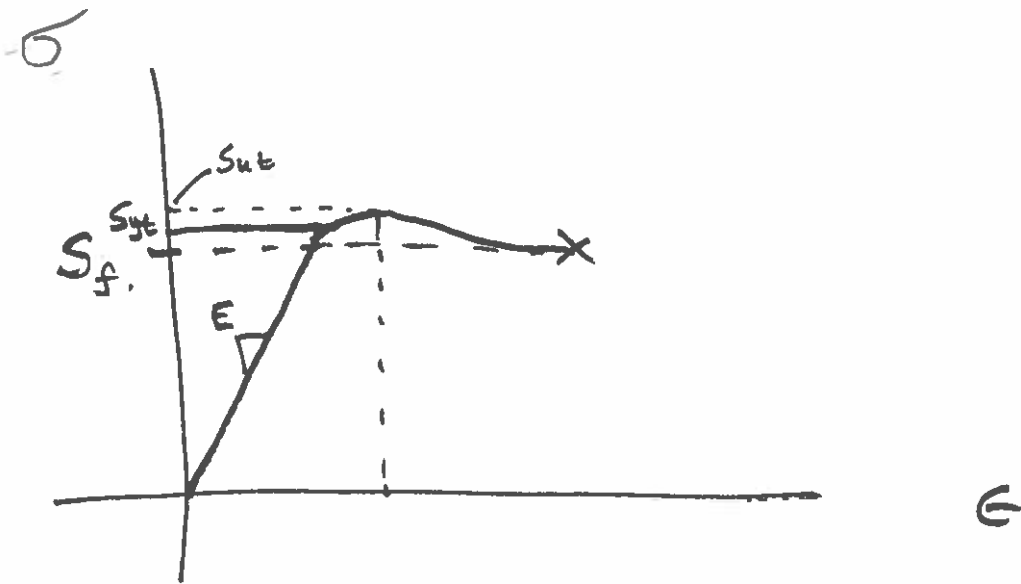
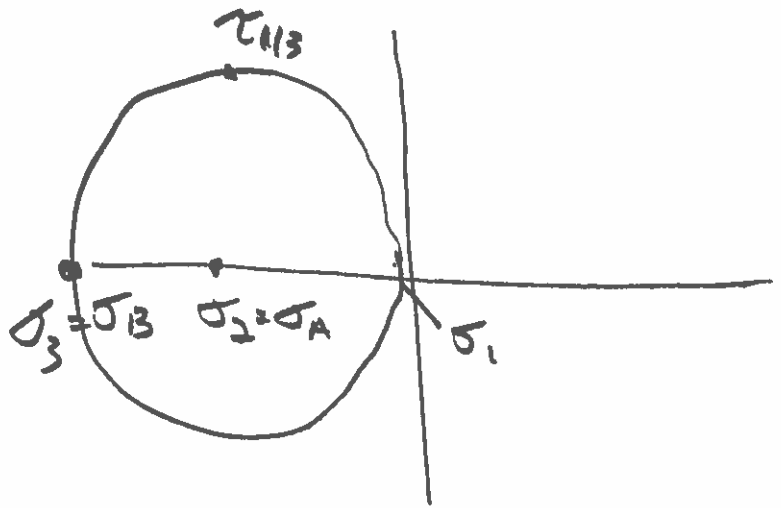
2) $\sigma_A > 0 > \sigma_B$

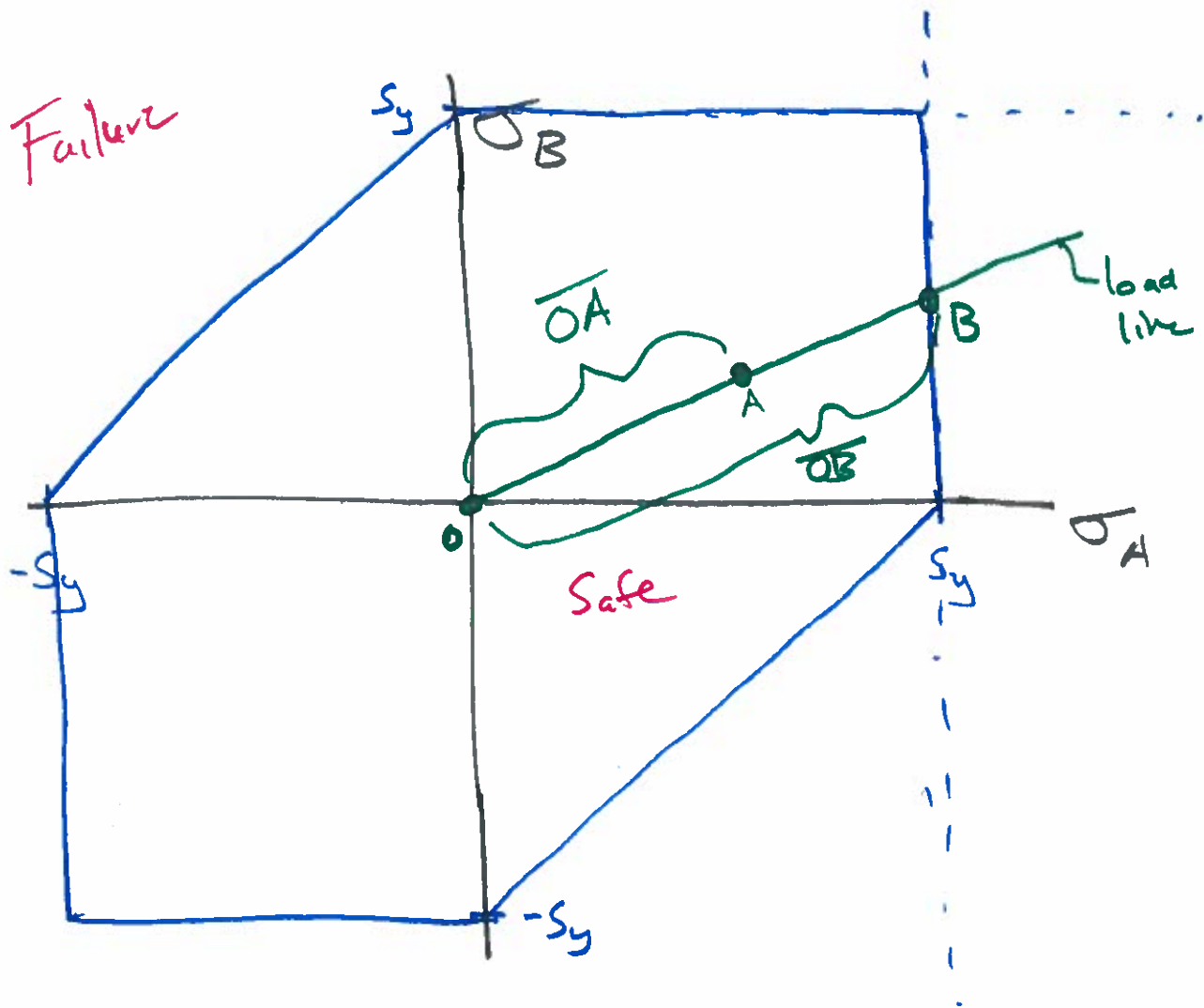
$$\sigma_A - \sigma_B \geq S_y$$



3) $0 > \sigma_A > \sigma_B$

$\sigma_B \leq -S_y$





$$n_{mss} = \frac{\overline{OB}}{\overline{OA}}$$

$$\sigma_A \geq \sigma_B$$

Distortion Energy Theory

$$U = \frac{1}{2} \epsilon \sigma$$

distortion strain energy
per unit volume

$$\left[\frac{(\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2}{2} \right]^{1/2} \geq S_y$$

σ' = Von mises stress

constitutes failure

σ'

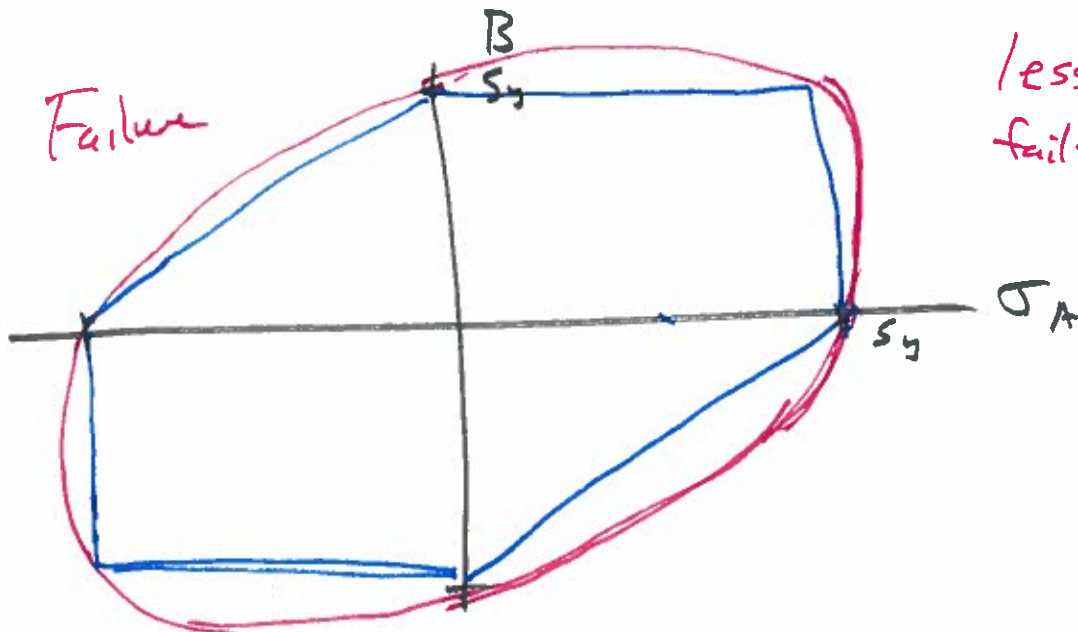
$$n_{de} = \frac{S_y}{\sigma'}$$

Plane Stress

$$\sigma_1 = \sigma_A$$

$$\sigma_2 = \sigma_B$$

$$\sigma' = (\sigma_A^2 - \sigma_A \sigma_B + \sigma_B^2)^{1/2}$$



less conservative
failure theory